Calculation of the Rosch Buoyancy Plant by A. Schneider

The buoyancy is caused by the cylindrial floats on one side of the paternoster, where compressed air is pumped in on the lowest point.

Jan 02, 2014/AS Jan 07, 2014/AS Jan 10, 2014/AS Jan 21, 2014/AS

Used constants

| Mathematical Constant Pi | π | 3.14159 |
|---|-------------------|---------------|
| Constant of Gravitation | g | 9.80620 m/s2 |
| Water density at 22 degree Celsius | ρ_{w} | 998.777 kg/m3 |
| Air density at 20 degress celsius and pressure of 1 bar | ρ_{A} | 1.29 kg/m3 |

| Force calculation from buyoncy | | |
|-----------------------------------|-------------|-----------|
| 1. Radius of first half cylinder | r_1 | 0.12 m |
| 1. Length of first half-cyinder | I_1 | 0.24 m |
| 1. Radius ofsecond half cylinder | r_2 | 0.12 m |
| 1. Length of second half-cyinder | I_2 | 0.64 m |
| 1. Volume of first cylinder | V_1 | 10.86 l |
| 2. Volumen of second cylinder | V_2 | 29.13 l |
| Both cylinder together = 1 float | $V=V_1+V_2$ | 39.99 I |
| Numer of floats on one side | Z | 15 |
| Mean water displacement per float | f | 50% |
| Total volume of suppressed water | $V_{\rm w}$ | 299.93 I |
| Total weight of suppressed water | m | 299.57 kg |
| Air weight in the buoyancy floats | G_L | 0.39 kg |
| Buoyancy force in Newton | K | 2'934 N |

Red: = Calculations made by Rosch AG see pdf-File "EN Rosch Power Plant-final"



| Force calculation | | | p. 4 |
|-------------------|----|----------|------|
| 299,63 | kg | 6th line | |
| 2'939 | N | 7th line | |

| Turns per min. of the upper paternoster wheel | n | 2 U/min. | | |
|---|-------------|---------------|----------------------|-------------|
| | = | 0.033 U/s | | |
| Diameter of the paternoster wheel | d | 0.5 m | = 500 mi | m |
| Velocity of the chain in m/min | ٧ | 3.14159 m/min | Traveled dist | ance |
| | ٧ | 0.05236 m/s | 0.052 | 4th line |
| Height of the chain with 15 floats | h | 8 m | | |
| Needed time for passing the height h | t | 152.79 s | | |
| Released energy over the height h | Е | 23'470.56 Nm | Thrust force | calculation |
| Generated power | $P_a = E/t$ | 0.154 kW | 0.154 | 4th line |



733.46 Nm $P_a = 2*\pi*M*n$ 0.154 kW corresponding to Rosch P_{in} = 0.149 kW 3% loss 5th line

Transmitted input power to the generator with 3% loss Required minimal diameter of axle of the chain wheel

Mechanical output power P calculated via torque and rpm

for haevily loaded short shaft with τ = 78 N/mm

Torque (Radius chain wheel *buoyancy force)

Generated power on the chain wheel

 $(d = 36.5 * (P/n)^{0.33} * (1/\tau)^{0.33})$ d = **36.31** mm corresponding to Rosch Reference: http://www.ignou.ac.in/upload/Unit-7-60 p.168f.

p.5

=1/13 wheel diameter = 500 mm/13=

standard size = 38 mm

the wheels shaft from photo

d diameter of

38.462

Energy (power) consumption of compressor

| Mean volume of compressed air over the heigt h | $V_{\rm w}$ | 299.93 I | Energy con | sumpt | ion | p.5 |
|--|---------------------------|---------------|-------------|-------|----------|------------|
| Correcton factor for decompressed air | k | 1.40 | calculation | | | |
| Effectively needed volume at norm pressure | V_{k} | 419.91 l | | | | |
| Needed air consumption in I/s | | 2.75 l/s | | | | |
| Needed compressor power minimal | | 164.90 l/min. | | | | |
| Used Compressor (it is sufficient) | | 200.00 l/min. | | | | |
| Nominal power of the used compressor | | 1.70 kW | | | | |
| Effectively needed power of the compressor | $\mathbf{P}_{\mathbf{e}}$ | 1.40 kW | 1,40 | kW | 9th line | |

Efficiency buoyancy plant

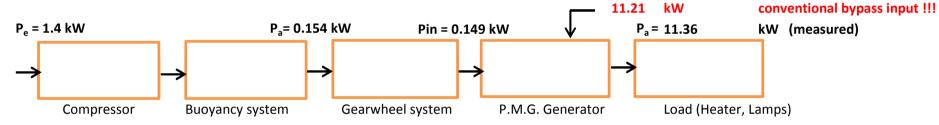
 P_e/P_a 10.96 %

Relationship of mechanical output to eletrical input

| Electrical output power generator (measured) | P_{out} | 11.36 kW | Nov. 15, 2013, DiplIng. Adolf Schneider |
|--|---------------------|--------------|---|
| | | 12,00 kW | Aug. 10, 2013, DiplPhys. DiplIng. W. Fack |
| Generator efficiency | | 98% | 2% loss 6th line p.5 |
| Mechanical power in from electrical output power | P_{m} | 11.59 kW | derived from the measurement of Nov. 15, 2013 |
| Transmission efficiency | | 97% | 3% loss 5th line p.5 |
| "Calculated mechanical power" of the paternoster | P _{PN} | 11.71 kW | This power cannot be present at the shaft of the upper |
| from the electrical output power | | | wheel of the paternoster because the steel of the shaft |
| Theoretical Torque on the wheel's shaft | $M = P_m/(2*\pi*n)$ | 16'510.06 Nm | cannot sustain the torque M as calculated (on the left) |
| Theoretical tangential force on the wheel | K = M/radius | 66'040.22 N | Also the force K is to high for the tooths of the wheel !!! |

Block diagram of the complete Buoyancy system

Additional mysterious power P_{add} flowing in via "space energy" or other more



In the case that the additional power is flowing into the generator via special designed magnetic system which converts some space energy via coupling of the elementary magnets to the Quantum Vacuum we can calculate a very high COP by relating the electrical output to the electrical input

Efficiency of the complete system

$$P_{a}/P_{e} = 810\%$$

Such an efficiency would allow that the required input power could be feeded back from the output power which is 8.1 times the input power. But this is only possible when the mysterious additional input power can be traced down (bypass path from grid or O/U-P.M.G. generator)

Standard P.M.G. generators: see e.g. products from Polen http://pmg-technology.com/o-firmie/ **Generators with COP >> 1:** see e.g. http://www.borderlands.de/Links/HighEfficientPMGenerators.pdf